

$$T = \frac{60}{L} J$$

$$\theta = \frac{TL}{JG}$$

$$L = 2 \text{ m}$$

$$T = 1 \text{ Nm}$$

$$J = 10^{-8} \text{ m}^4$$

$$G = 80 \text{ N/m}^2$$

1800 rpm

rad/s = $\frac{2\pi}{360}$ degree

$$\theta = 57.3 \text{ rad}$$

substiti $\frac{T}{J} = \frac{60}{L}$ from eqn (1) $\left(\frac{T}{J} = \frac{60}{L}\right)$

$$T = \frac{T}{J} \rho$$

$\rho \rightarrow$ material r

$$\max T = \frac{T \cdot r}{J}$$

$$\max \tau = \frac{T \cdot \frac{D}{2}}{\frac{\pi D^4}{32}} = \frac{16T}{\pi D^3}$$

shear stress

$$= \frac{16TD}{\pi(D^4 - d^4)}$$

hollow shaft

shear transmitted power

$$P = T \cdot \omega$$

ω : radian/s

$$\omega = 2\pi f$$

f : frequency revolve / time

$$P = 2\pi f \cdot T$$

P : watt = N.m/s

f : 1/s, T : N.m

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Solid shaft

- $P = 20 \text{ kW}$
- $f = 20 / \text{s}$
- $d = ?$
- $T_{\text{max}} = 40 \text{ MN/m}^2$
- $\theta = 6^\circ$
- $L = 3 \text{ m}$
- $G = 83 \text{ GN/m}^2$

Soln:

$$P = T \cdot \omega$$

$$= T \cdot 2\pi f$$

$$T = \frac{P}{2\pi f} = \frac{20 \times 10^3}{2\pi \times 20} = 1590 \text{ N}\cdot\text{m}$$

$$T = \frac{16T}{\pi d^3} \quad / \quad 40 \times 10^6 = \frac{16 \times 1590}{\pi d^3}$$

$$d = 58.7 \text{ mm}$$

$$\theta = \frac{T \cdot L}{GJ}$$

$$6 \times \frac{1}{57.3} = \frac{1590 \times 3}{83 \times 10^9 \times J} \left(\frac{\pi}{32} d^4 \right)$$

$$d = 48.6 \text{ mm}$$

$$\therefore d = 58.7 \text{ mm}$$